

HPC enabling of OpenFOAM[®] for CFD applications

Numerical acoustic analysis of a turbulent flow around a bluff body

06-08 April 2016, Casalecchio di Reno, BOLOGNA.

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- Motivations
- Overview on Aeroacoustics
- Numerical Approach
- Case of Study
- Future steps

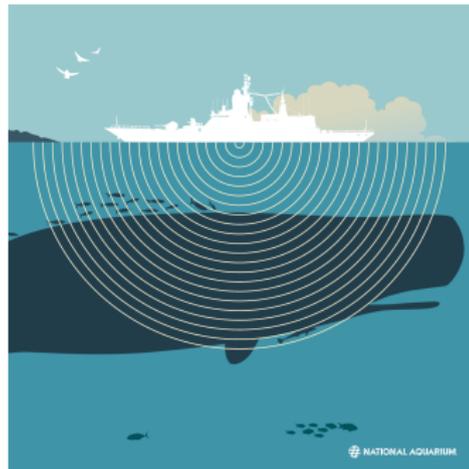
Aeroacoustic study of fluid-mechanically-generated sound has become an important research endeavor because of the growing need to control transport noise.

As ship design advances, particularly with regard to structural optimisation and high speeds to meet market demands, there is a tendency for **noise and vibration problems** to become more pronounced.

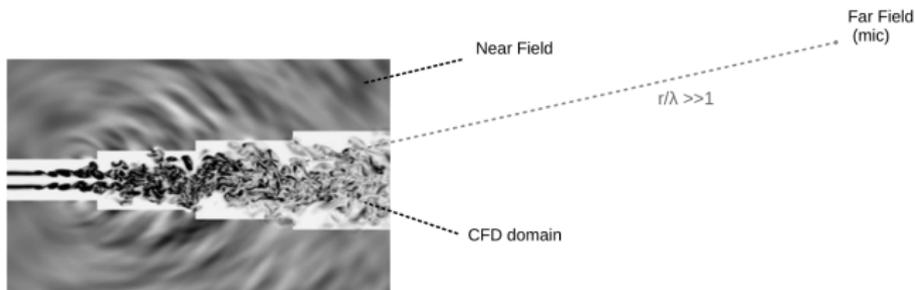
And it follows an increasing attention on **acoustic pollution** and its impact on marine life.

Ship underwater noise is quite unexplored research field: when looking at the literature, it is easy to recognize a **lack of both theoretical and computational models**. At present, the criteria adopted to satisfy the noise emission requirements are based on empirical basis and the use of some approximated numerical procedure.

The hydroacoustic behaviour of a marine propeller is generally not investigated at design stage.



Direct vs Hybrid methods



Free jet noise, N. Sandham, Z. Hu and C. Morfey

Acoustic Scales

Speed of sound c : air 340 m/s; water 1480 m/s

Human reception of frequencies f : 20 Hz - 20 kHz

Wavelength $\lambda = c/f$ (es. on air 17mm-17m)

Lighthill 1952

The **Lighthill equation** represent a rearrangement of the fundamental conservation laws of mass and momentum into an inhomogeneous wave equation:

$$\left\{ \begin{array}{l} \frac{\partial p}{\partial t} + \nabla(\rho u_i) = 0 \\ \frac{\partial}{\partial t}(\rho u_i) + \nabla(\rho u_i u_j + p \delta_{ij} - \tau_{ij}) = 0 \end{array} \right. \implies \square^2 p = \underbrace{\frac{\partial^2}{\partial x_i \partial x_j} (\rho u_i u_j - \tau_{ij} + (p - c_0^2 \rho) \delta_{ij})}_{\text{Acoustic source}}$$

$$c_0^2 = \left(\frac{\partial p}{\partial \rho} \right)_{s_0}$$

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The RHS term includes all possible noise source mechanisms taking place in the flow:

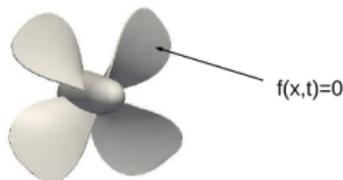
- $\square^2 = \left(\frac{1}{c_0^2} \frac{\partial^2}{\partial t^2} - \nabla^2 \right)$
- the convective term, represented by Reynolds tensor $\rho u_i u_j$
- the possible deviation from isentropic behaviour $(p - c_0^2 \rho) \delta_{ij}$
- viscous stresses τ_{ij}

Ffowcs-Williams, Hawkins 1967

The **Ffowcs Williams-Hawkins equation** is an extension of Lighthill work, accounting for the possible presence of a body moving in the fluid.

Such a presence is described by representing the surface $f(x, t) = 0$ (on which $u_n = v_n$) as a moving discontinuity in the flow and, then, re-writing the same conservation laws in terms of generalized functions:

$$\square^2 p H(f) = \underbrace{\frac{\partial^2}{\partial x_i \partial x_j} \{T_{ij} H(f)\}}_{\text{Vol Quadrupole source}} + \underbrace{\frac{\partial}{\partial t} \left(\rho_0 v_i \frac{\partial f}{\partial x_j} \delta(f) \right)}_{\text{Surf Monopole source}} - \underbrace{\frac{\partial}{\partial x_i} \left(P_{ij} \frac{\partial f}{\partial x_j} \delta(f) \right)}_{\text{Surf Dipole source}}$$



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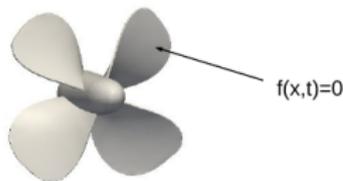
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Where:

- $T_{ij} = \frac{\partial^2}{\partial x_i \partial x_j} (\rho u_i u_j - \tau_{ij} + (p - c_0^2 \rho) \delta_{ij})$
- $P_{ij} = \rho \delta_{ij} - \tau_{ij} - \rho_0 \delta_{ij}$

Compared to Lighthill equation and the original flow 3D term, two additional surface 2D terms appear, known as **thickness** and **loading** noise components.



Previous equations are turned into an **integral form** by using the free-space Green function for the wave equation:

$$G(x, y; t, \tau) = \frac{\delta(t - \tau - |x - y|/c_0)}{4\pi|x - y|}$$

The solution of an inhomogeneous wave equation of the kind:

$$\square^2 p = q(x, t) |\nabla f(x)| \delta(f)$$

is given by the following, integral form:

$$p(x, t) = \int_{-\infty}^t \int_V G(x, y; t, \tau) q(y, \tau) |\nabla f(x)| \delta(f) dV(y) d\tau$$

The emission time $\tau = t - |x - y|/c_0$ (at which the integral has to be computed) represents the instant at which the noise is emitted from y to reach the observer x at time t .

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We get to the **Formulation 1**, originally developed for aeroacoustic analysis of helicopter rotors, refers to FWH **2D terms**, which depend on body shape (v_n , thickness noise) and surface loads (p , loading noise)

$$4\pi p(x, t) = \frac{\partial}{\partial t} \int_S \left[\frac{\rho_0 v_n}{r|1 - M_r|} \right]_{\tau} dS + \frac{1}{c_0} \frac{\partial}{\partial t} \int_S \left[\frac{p r_n}{r|1 - M_r|} \right]_{\tau} dS + \int_S \left[\frac{p r_n}{r^2|1 - M_r|} \right]_{\tau} dS$$

With $M_r = v \cdot (x - y)/c_0$.

The same Green function approach allows us to turn the nonlinear, Lighthill term into some **3D integrals**, where the integration domain theoretically represents the whole volume of fluid V affected by body motion, but it turns out to be CPU time demanding.

In 1997 the first numerical results coming from the **FWH porous formulation** were published. This formulation computes the FWH integrals on a closed domain S_p , outer the body source and permeable ($u_n \neq v_n$), which embeds possible nonlinear sources.

$$4\pi p(x, t) = \frac{\partial}{\partial t} \int_S \left[\frac{\rho_0 U_n}{r|1 - M_r|} \right]_{\tau} dS + \frac{1}{c_0} \frac{\partial}{\partial t} \int_S \left[\frac{L_r}{r|1 - M_r|} \right]_{\tau} dS + \int_S \left[\frac{L_r}{r^2|1 - M_r|} \right]_{\tau} dS$$

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Where:

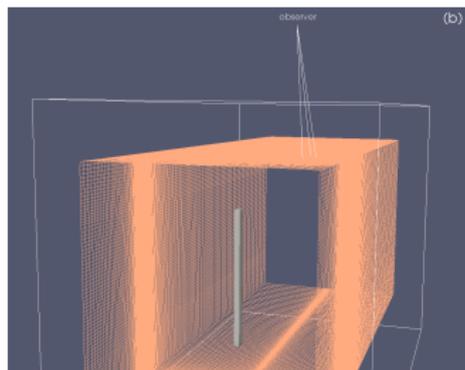
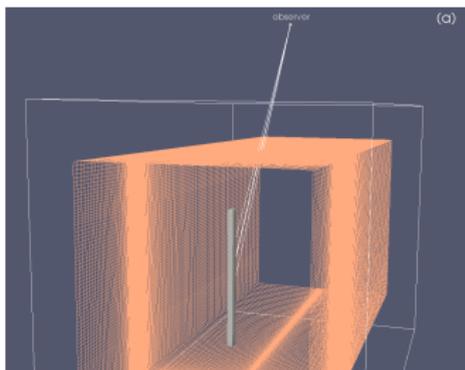
- $U_i = \left(1 - \frac{\rho}{\rho_0}\right) v_i + \frac{\rho}{\rho_0} u_i$
- $L_i = P_{ij} n_j + \rho u_i (u_n - v_n)$

By suitably fixing S_p , the porous equation (automatically accounting for the contribution of the nonlinear sources) can avoid any volume integration. However, an accurate solution of the fluid-dynamic problem on S_p has to be available.



S. Ianniello, The Ffowcs Williams-Hawkins equation for hydroacoustic analysis of rotating bodies

In order to implement formulation 1 (a) or the porous formulation (b) we need to collect the data either on the body surface or on the porous domain.



We need to know for **each cell** of the “source domain” (at **each time step**) : the flow data (**p** and **U**), the outward normal to the surface **n**, the distance from the observer **r**, the area of the cell **dS**.
 Then we integrate on the surface, and do derivative in time for some terms.

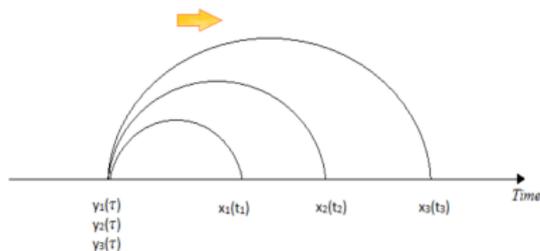
Numerical Approach

The time τ at which the integral has to be computed is the **emission time**, it represents the instant at which the noise is emitted from y to reach the observer x at time t .

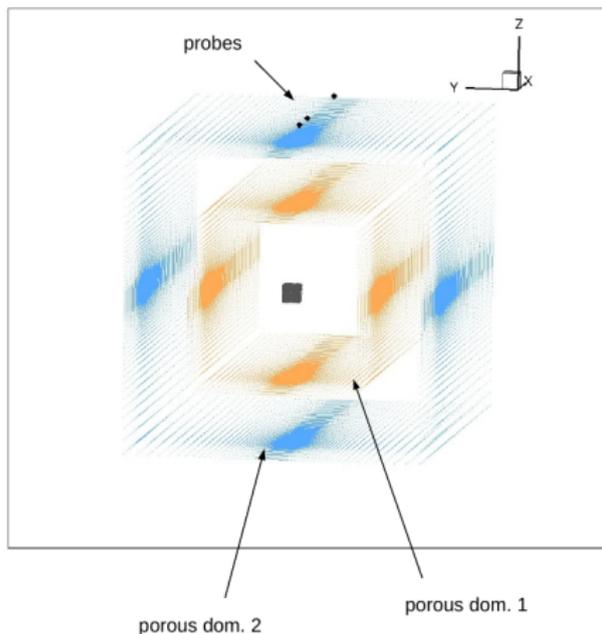
$$\tau = t - \frac{|x - y|}{c_0}$$

The difference between τ and t is a fundamental feature of propagation mechanisms: it is known as compressibility delay and points out that the speed of sound c_0 is finite.

We can fix τ and move forward in time in order to determine t . **Data-fitting** is then required to rebuild the overall, resulting noise $p(x, t)$.



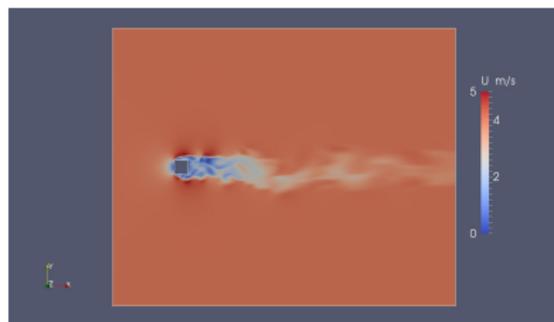
Noise of a subsonic flow around a **cube**



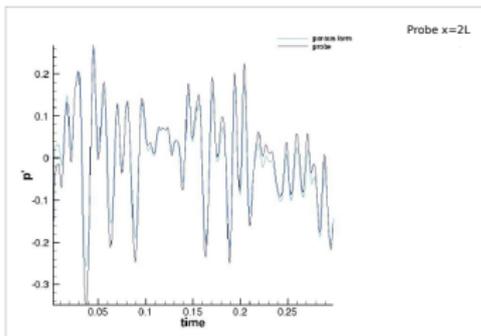
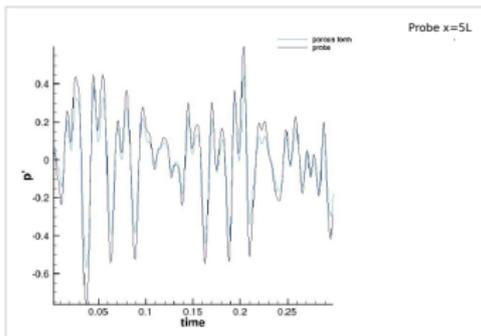
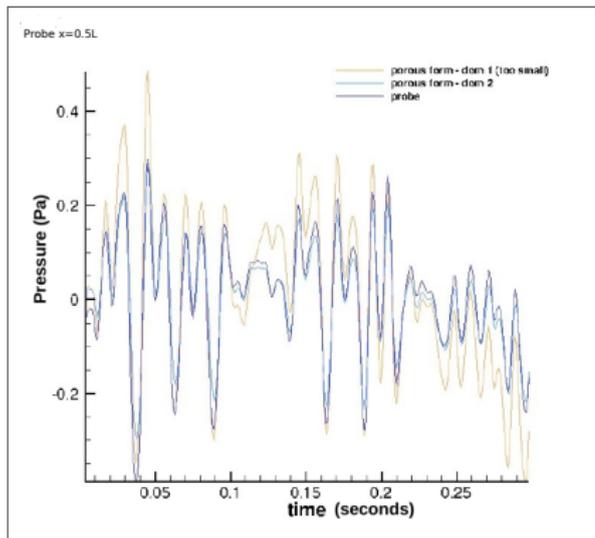
$$U_{in} = 4\text{ m/s}, \quad L = 0.02\text{ m}$$

$$Re_D = 8000$$

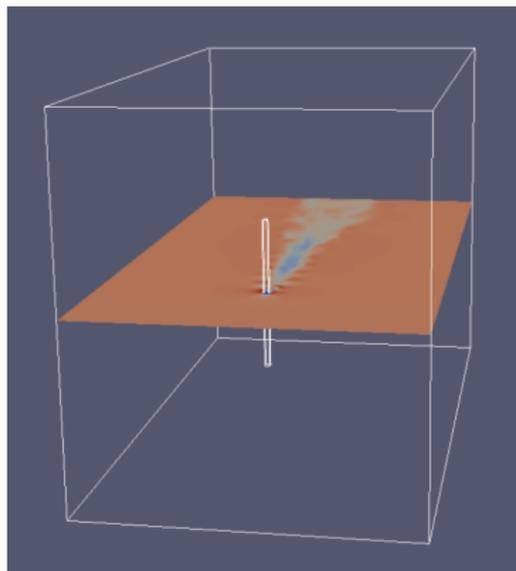
The cube is situated in a tunnel $20L$ width, $20L$ high and $35L$ long, cyclic boundary conditions are imposed on the sides of the tunnel.



Noise of a subsonic flow around a cube



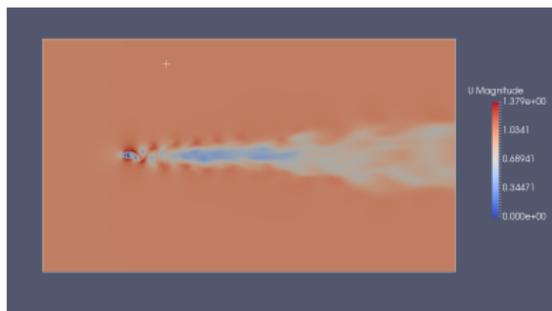
Noise of a subsonic flow around a square cylinder

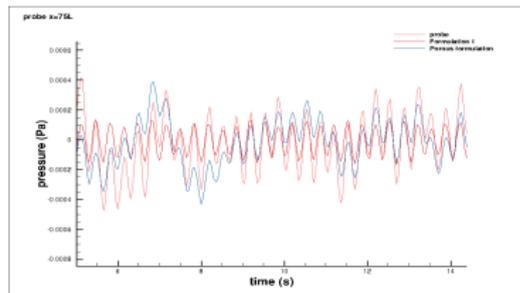
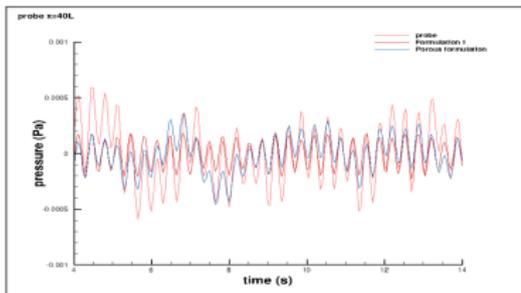
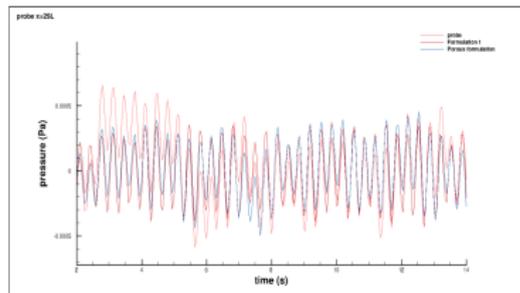
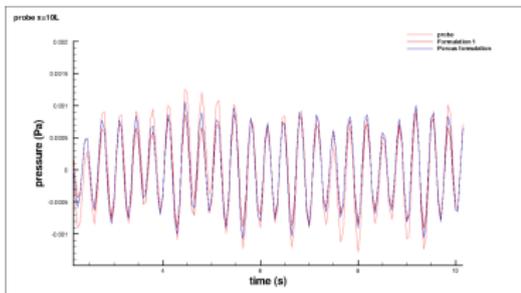


$$U_{in} = 1.2m/s, \quad L = 0.05m, \quad H = 1.2m$$

$$Re_D = 6000$$

The square cylinder is situated in a tunnel 70L width, 80L high and 125L long, cyclic boundary conditions are imposed on the sides of the tunnel.





- Study how the 3D terms affect the far field noise

$$\begin{aligned}
 4\pi p_{NL}(x, t) = & \frac{1}{c_0^2} \frac{\partial^2}{\partial t^2} \int_V \left[\frac{T_{rr}}{r|1 - M_r|} \right]_{\tau} dV \\
 & + \frac{1}{c_0} \frac{\partial}{\partial t} \int_V \left[\frac{3T_{rr} - T_{ii}}{r^2|1 - M_r|} \right]_{\tau} dV \\
 & + \int_V \left[\frac{3T_{rr} - T_{ii}}{r^3|1 - M_r|} \right]_{\tau} dV
 \end{aligned}$$

- Try a more complex configuration, such as a system of several cubes or a flow passing through a grid
- Learn how to treat with moving boundaries, in order to study the underwater propeller noise production

- ◇ Computational Prediction of Flow-Generated Sound, M.Wnag, J.Freund, S.Lele, Annu.Rev.Fluid.Mech., 2006.
- ◇ Recent Advances of Computational Aeroacoustics, X.Li et al., Appl.Math.Mech., 2015.
- ◇ On Sound Generated Aerodynamically I General Theory, M.Lighthill, P Roy Soc Lond A, 1952.
- ◇ Sound Generation by Turbulence and Surfaces in Arbitrary Motion, Ffowcs, Hawkings, Phys.Sciences, 1969.
- ◇ An Introduction to Aeroacoustics, S.W.Rienstra, A.Hirschberg, 2004.
- ◇ Basics of Aeroacoustics, Lecture Notes of Prof. J.Delfs, 2014.
- ◇ Large-Eddy Simulation for Acoustic, C.Wagner, T.Hutti, P.Sagaut, Cambridge University Press, 2007.
- ◇ Computational Aeroacoustics of Complex Flows at Low Mach Number, thesis of Y.Khalighi, 2010.
- ◇ Computation of Trailing Edge Noise from an Incompressible Flow Calculation, Oberai et al., Cent.Turb.Research, 2000.
- ◇ Computation of Trailing Edge Noise Due to Turbulent Flow over an Airfoil, Oberai et al., AIAA Journal, 2002.
- ◇ A Hybrid LES/CAA Method for Aeroacoustic Applications, Zhang et al., High Performance Computing on Vector Systems, pp 139-153, 2005.
- ◇ High Performance Computing on Vector Systems, Resch et al., Springer ed., 2005.
- ◇ Wave Packets and Turbulent Jet Noise, P.Jordan, T.Colonius, Ann.Rev.Fluid Mech., 2013.
- ◇ Acoustic Analogy Formulations Accelerated by Fast Multipole Method for 2D Aeroacoustic Problems, R.Wolf.,K.Lele, AIAA Journal, 2010.
- ◇ Implementation of Acoustic Analogies in OpenFoam for Computation of Sound Fields, J. Schmalz, W. Kowalczyk, Open Journal of Acoustics, 2015 .
- ◇ Flow-induced Noise Analysis for 3D trash rack based on LES/Lighthill hybrid method, Y.Zhang, H.Ouyang, Applied Acoustic, 2014.

Thank You

