







Numerical simulation of the Mont Blanc Tunnel ventilation system

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Overview The TMB and its ventilation system Experimental work Numerical work

Outline



- Up to full size
- 3 Implementation
- Results of pre-production runs
- 5 Conclusions and further work

Where it all began...

Collaboration started in 2009, among:

- Department of Engineering Enzo Ferrari (DIEF), University of Modena and Reggio Emilia
- Gruppo Europeo di Interesse Economico del Traforo del Monte Bianco (GEIE-TMB)
- mimesis s.r.l., a spin-off of DIEF

on the study and optimization of the Mont Blanc tunnel ventilation system.

Project objectives:

- to collect extensive physical data and information on the airflow in the tunnel
- to develop predictive tools for the study of ventilation configurations and/or critical scenarios
- to propose new control strategies to reduce the response times in case of event

Methodologies:

- The research is carried out by both modeling and experiments, combined in an integrated approach for analysis of this class of problems
- Experiments provided accurate in situ air velocity data for model development and validation, and the verification of airflow control infrastructures

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The TMB and its ventilation system I

The Mont Blanc Tunnel:

Two-way road tunnel connecting Italy and France, **11611 meters** long with three different slopes

The tunnel is equipped with a hybrid semi-transverse ventilation system, including:

- A fresh air supply system, with 1160 vents along the Italy-France sidewalk
- A smoke extraction system, with 116 openings on the tunnel vault
- A longitudinal flow control system, with 38 jet fan couples mounted on the tunnel ceiling
- A pressurization system for the stashes (recovery in event of fire)
- A number of sensors (anemometers, barometers, thermometers, pollutants detectors, differential pressure sensors)

Overview **The TMB and its ventilation system** Experimental work Numerical work

The TMB and its ventilation system II



Tunnel schematic

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Experimental work

In vivo experimental campaigns

- Fixed-point measurements
- Continuous profile acquisitions (T.A.L.P.A.)

Jet fan characterization

- static profiles
- hot wire anemometry
- measurement of u_z , u_r , u_{θ} , k



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Objectives of the numerical work

 Development of a 1D simplified network model to simulate the global behaviour of the ventilation system

Development of a complete CFD model of the whole tunnel

- ... to enable detailed flow analyses on a global scale
- ... to demonstrate the feasibility of CFD for such a large and multi-scale domain
- ... to provide the basis for future studies of real scenarios (air quality control with heavy traffic loads, fire events, etc...)



Critical aspects Modeling choices Physical equivalents Validation runs

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Critical aspects

- Huge domain: an evident computational challenge
- Ventilation elements: jet fans, fresh air vents, smoke extraction outlets imply the presence of jets and shear layers of different scales and orientations
- Complex modularity: the automation of the geometrical modeling and discretization phase is hindered by the non-trivial layout of the different ventilation elements (see below)
- Presence of small scale geometrical details such as sidewalks, niches, lighting, panels, sensors, etc. which have an influence on the flow



Modeling choices I

- **Turbulence**: the scale of the problem makes resolved turbulent simulations like LES unfeasible. Turbulence is accounted for by means of the **realizable** $k \varepsilon$ RANS model
- Geometrical simplifications: small scale details are not included in the models, and are substituted by an equivalent wall roughness
- Jet fan modeling: jet fans are modeled as hollow cylinders with ad-hoc BCs (swak4Foam) at the intake and discharge sections
- Lateral air supply vents: the accurate modeling and meshing of the small air supply vents is regarded as a computational overkill. The vents have been substituted by continuous boundary patches in the computational model
- All physical equivalences have been suitably tuned thanks to the extensive experimental database

Critical aspects Modeling choices Physical equivalents Validation runs

Modeling choices II



Critical aspects Modeling choices Physical equivalents Validation runs

Wall roughness

The friction factor *f* is extracted from experimental data using the Darcy-Weisbach equation:

$$\Delta p = f \frac{L}{D_h} \frac{\rho U^2}{2}$$

The average surface roughness ε is calculated via the Colebrook equation:

$$\frac{1}{\sqrt{f}} = -2\log_{10}\left(\frac{\varepsilon}{3.7D_h} + \frac{2.51}{Re\sqrt{f}}\right)$$

The logarithmic law of the wall for hydraulically rough flows is then enforced:

$$u(y) = \frac{u_{\tau}}{\kappa} \ln \frac{y}{y_0}$$
 $y_0 = \frac{\varepsilon}{30}$

Critical aspects Modeling choices Physical equivalents Validation runs

Jet fan ad hoc BCs

At the **fan suction inlet** (an outlet for the CFD model):

- pressure is imposed as equal to the average pressure at the discharge section, minus the nominal pressure jump given by the fan;
- a zero gradient condition is imposed for velocity components.

$$p_s = \bar{p}_d - \Delta p_{fan} \qquad \frac{\partial \mathbf{u}}{\partial n} = 0 \tag{1}$$

At the **fan discharge** (an inlet for the CFD model):

- velocity is imposed as equal to the average velocity at the suction, multiplied by the reconstructed dimensionless velocity profile (ensuring mass conservation);
- a zero gradient condition is imposed on pressure.

$$\mathbf{u}_{d} = \bar{U}_{s} \,\tilde{\mathbf{u}}_{\mathbf{d}} \qquad \frac{\partial p}{\partial n} = 0 \tag{2}$$

Validation runs I

Simulations with 1,2 and 3 active jet fans couple across a 900 m long segment of the tunnel compared with experimental data in order to test physical equivalences

- isothermal and steady-state
- unidirectional flow field
- upstream and downstream distributed pressure losses taken into account at domain's end (with swak4Foam)
- second order spatial discretisation schemes



Validation runs II

Good agreement in terms of longitudinal mean velocity

N. of active fan couples	1	2	3
U _{exp} [m/s]	3.10 ± 0.20	3.21 ± 0.20	3.17 ± 0.20
U _{num} [m/s]	3.209	3.387	3.399

Reliable simulation of swirl component in fan discharge section



The project Toolchain Up to full size Modeling strategy Implementation Assembly of the whole model Results of pre-production runs Scalability Conclusions and further work Reconstruction and post-processing

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Toolchain

The work has been carried out entirely with open source software:

- OpenFOAM[®]: pre-processing and solving phases
- **PyFoam**: exploited to chain the scripts used throughout.
- swak4foam: fast and easy implementation of customized BCs
- ParaView: post-processing

Toolchain:

- gnu/4.5.2
- python/2.7
- openfoam/2.1.1-gnu-4.7.2
- pyfoam/0.5.7
- swak4foam0.2.1
- ParaView/3.14

Toolchain Modeling strategy Assembly of the whole model Scalability Reconstruction and post-processing

Modular approach

- Exploitation of layout patterns repetition in the tunnel
- Seven elemental blocks, each 50 m long, have been singled out, which, if appropriately concatenated, could virtually represent the physical domain in its entirety*
- Meshing has been performed independently on each of the seven blocks using snappyHexMesh

*with some approximations



Toolchain Modeling strategy Assembly of the whole model Scalability Reconstruction and post-processing

Meshing I



Meshing II

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Assembly of the whole model

- Python/PyFoam scripts
- mergeMeshes utility (serial process)
- Arbitrary Mesh Interface (AMI) to connect faces of consecutive blocks
- Different slopes taken into account
- Boundary conditions assignment
- The procedure is general and absolutely portable
- Final model assembly @ CINECA
 - 138 M of cells
 - Usage of 91 GB of RAM:
 - a "fat node" was used
 - 3 hours of wall-time



Toolchain Modeling strategy Assembly of the whole model Scalability Reconstruction and post-processing

Scalability

- Runs have been done on the compute nodes of the PLX cluster (2 hex-core Intel Xeon 2.40 GHz)
- buoyantBoussinesqSimpleFoam
- Three runs have been done using 12, 16 and 22 nodes, that correspond to 144, 192 and 264 cores



n. of nodes	n. of cores	cells per processor	n. of it	WCT	Cpu-hours
12	144	0.9 M	210	50812 s = 14.11 h	2032
16	192	0.7 M	210	39906 s = 11.09 h	2128
22	264	0.5 M	210	29126 s = 8.09 h	2135

Toolchain Modeling strategy Assembly of the whole model Scalability Reconstruction and post-processing

Reconstruction and post-processing

- The variable fields have been saved every 50 iterations, and they have been reconstructed using the **reconstructPar** utility.
- Memory demanding operation. Example: with Np=144 procs, 110 GB of memory used, 11 hours of cpu-time, and 13 GB of data occupancy for each time-field.
- Remote Visualization using ParaView on CINECA servers. It is mandatory, at least, to employ a node with 128 GB of memory or bigger.
- Loading time for the entire model is around 30 min. Simple **manipulation operations** can easily be performed.

TestCase number 1 TestCase number 2 Scalability Some snapshots

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TestCase number 1 TestCase number 2 Scalability Some snapshots

TestCase number 1

Based on fixed point measurements

- Ventilation system configuration:
 - Lateral air intake ON
 - Exhaust air extraction OFF
 - No. active jet fans NONE
- Run on 192 cores @ CINECA
- 800 iterations



TestCase number 1 TestCase number 2 Scalability Some snapshots

Results

Average longitudinal velocity: acceptable results

- Convergence not reached
- Uncertainty on measurements that reflects on BC
 - Lateral mass flow rate



TestCase number 1 TestCase number 2 Scalability Some snapshots

TestCase number 2

- Based on continuous measurements
- Ventilation system configuration:
 - Lateral air intake OFF
 - Exhaust air extraction ON from PM 9000 to PM 9600
 - No. active jet fans 4
 - 2: Italy→France
 - $\textbf{2:} \ \textbf{France} {\rightarrow} \textbf{Italy}$
- Run on 264 cores @ CINECA
- 700 iterations



TestCase number 1 TestCase number 2 Scalability Some snapshots

Results

Average longitudinal velocity: acceptable results

- Convergence not reached
- Uncertainty on experimental data
 - Strange loss of velocity, probably due to leakage before PM 9000 along the depressurized extraction channel
 - In the simulation extraction gates are perfectly closed



TestCase number 1 TestCase number 2 Scalability Some snapshots

Scalability

Simulation	n. of cores	n. of it	WCT	Cpu-hours
TestCase n. 1	192	700	61570 s = 17.10 h	3284
TestCase n. 2	264	700	48213 s = 13.39 h	3536

- Scalability on 700 iterations
- From 192 to 264 processors the scalability is around 80%

TestCase number 1 TestCase number 2 Scalability Some snapshots

Pre-production runs: some nice pictures I

Fresh air intake (TestCase n. 1):



TestCase number 1 TestCase number 2 Scalability Some snapshots

Pre-production runs: some nice pictures II

Counterflow jet fans (TestCase n. 2):

TestCase number 1 TestCase number 2 Scalability Some snapshots

Pre-production runs: some nice pictures III

Exhaust air extraction section (TestCase n. 2):

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Conclusions and further work

- "HPC enabling" of OpenFOAM®
- Satisfactory preliminary results
- Longer runs, more iterations for more accurate results
- Coupling modular 3D and 1D in a flexible multi-scale simulation

Contacts

Thanks you for your attention!

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