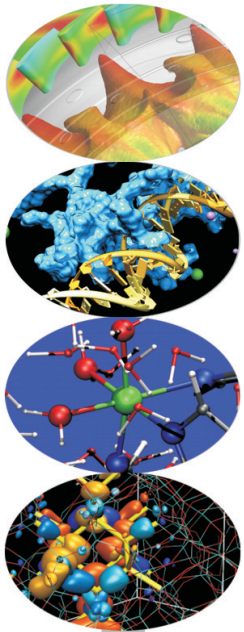


# Marine CFD applications using OpenFOAM

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27/03/2014



# Contents

- **Background** at CINECA: LRC experience
- **CFD** skills
- **Automatic** workflow
- **Reliability** workflow

# OpenFOAM solvers for marine CFD analysis

- **6DOF/2DOF solver:**  
*interDyMFoam* (dynamics, transient, optional wave motion)  
fully explicit mules: CFL mandatory
- **Unsteady 0DOF:**  
*interFoam* (transient captive)
- **0DOF (captive):**  
*LTSInterFoam* (Local Time Stepping (quasi-static hypothesis),  
suitable for automation and large computational campaign)

# OpenFOAM: CFD mandatory

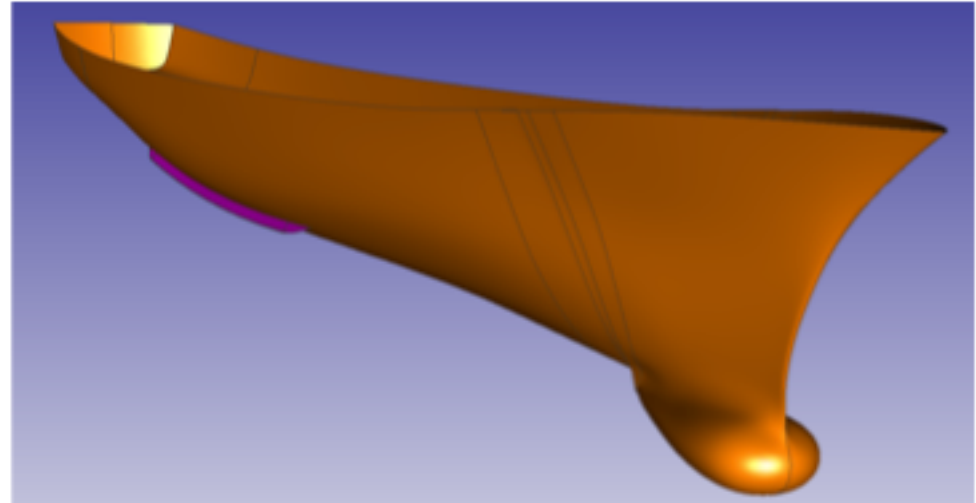
- OpenFOAM multiphase unsteady solvers have to respect the CFL condition:

$$\frac{u \Delta t}{\Delta x} \leq CFL = 1$$

- AC72 class: high speed (u), sufficiently small  $y^+$  → **too small time-step**
- **Commercial** softwares can manage 100-1000 times larger dt
- **Unsteady** simulation in OpenFOAM results too time consuming at the moment, but used only if mandatory due to the physics of the problem
- **LTSInterFoam**: local time stepping multiphase solver, developed "ad hoc" for Marine CFD.

# CFD Model

- ① High Reynolds simulation: **RANS** model employed
- ② Turbulence model: **k- $\omega$  SST**
- ③ **Wallfunction** enabled:  
 $y^+ \approx 70$



➤ *Standard DTMB-5415  
bare hull modeled*

# CFD model for marine applications

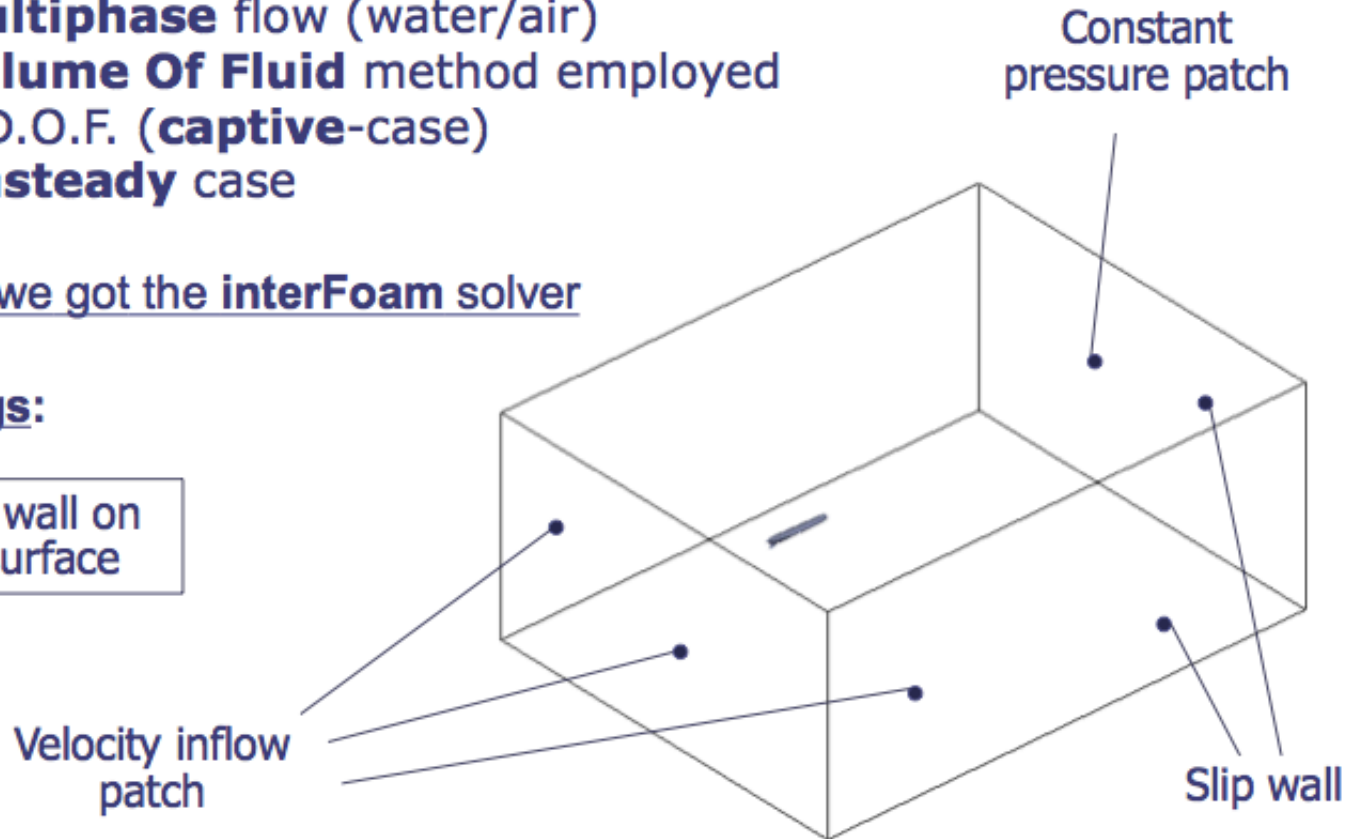
## 1. Solver:

- **Multiphase** flow (water/air)
- **Volume Of Fluid** method employed
- 0 D.O.F. (**captive-case**)
- **Unsteady** case

In OpenFOAM we got the interFoam solver

## 2. BC settings:

No slip wall on  
boat surface



# CFD comparison method

## 1. Qualitative:

- Iso-surface of computed mass-fraction
- Pressure on hull surface

Information about **wave shape, flow separation, stress distribution** on hull

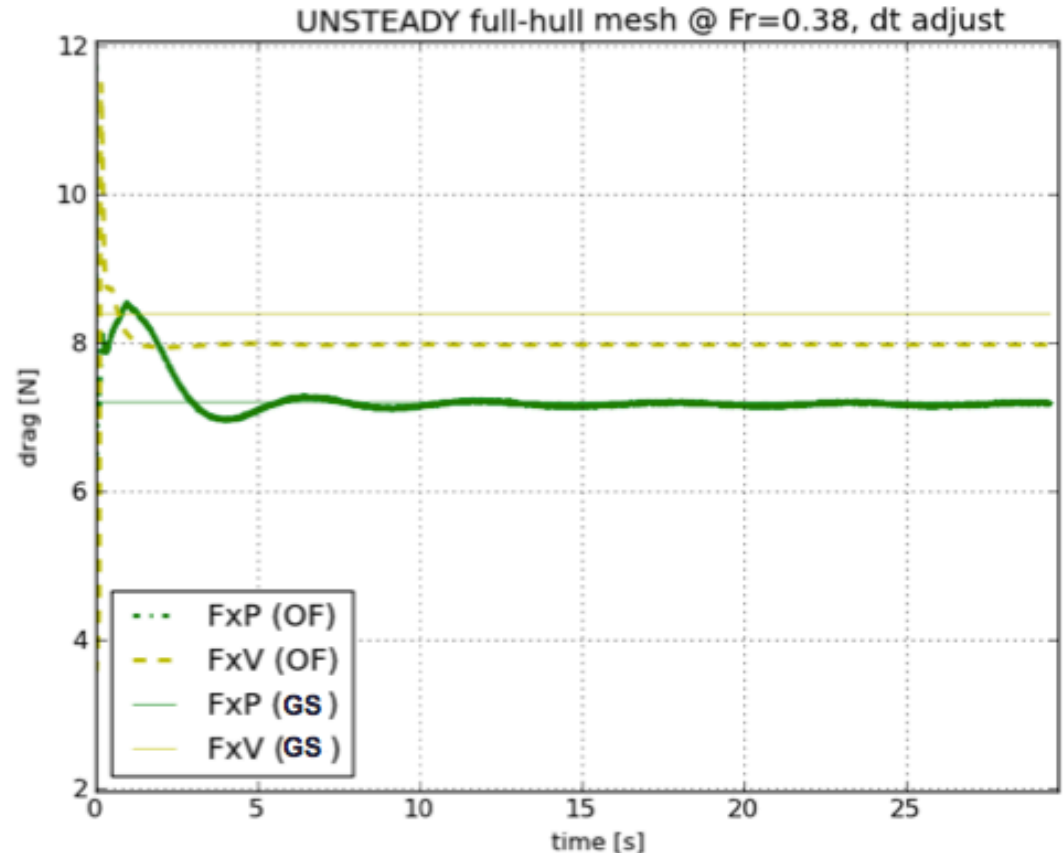
## 2. Quantitative:

- Pressure drag
- Viscous drag

My comparison is: OF vs GS (numerical)

# Unsteady captive - CFD Results

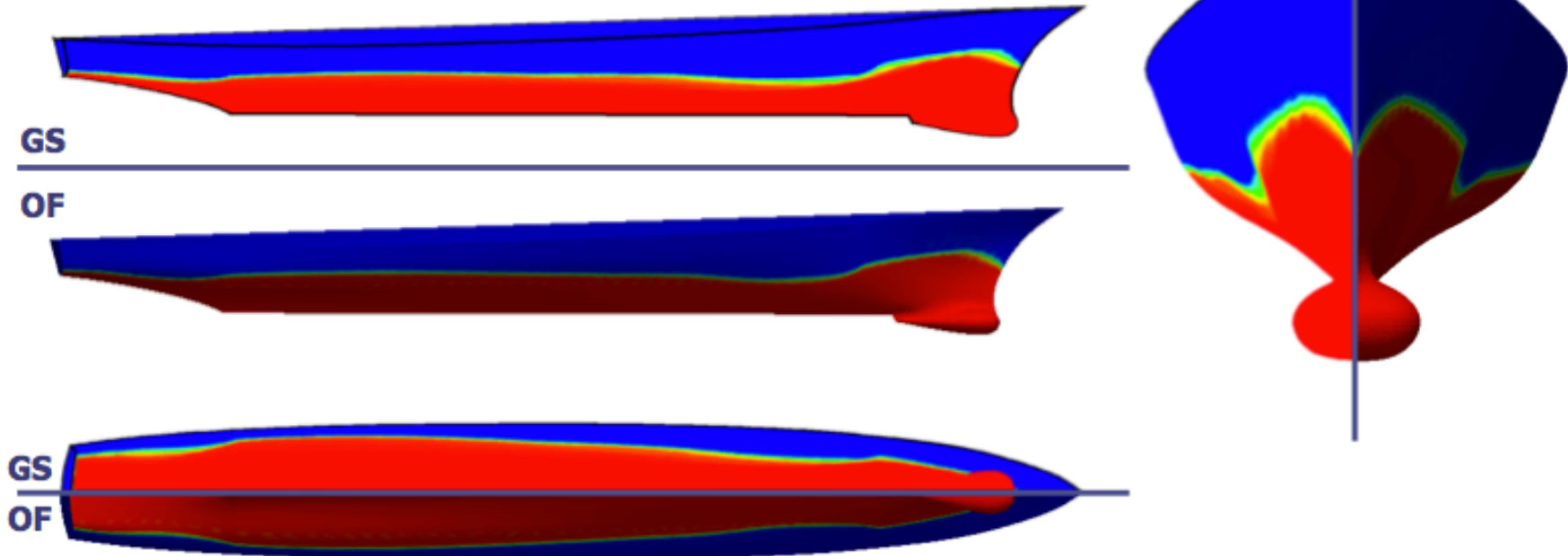
- Convergence reached in 10 s
- OpenFOAM vs Gold-Standard Drag values :
  - FxP: -0.34 %
  - FxV: -4.95 %
- Quite good agreement of results



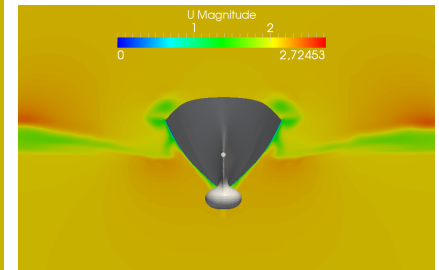
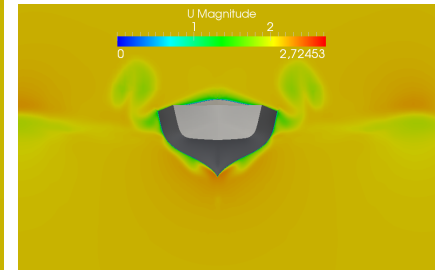
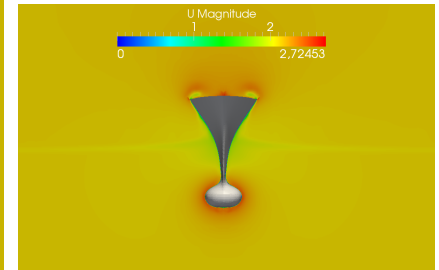
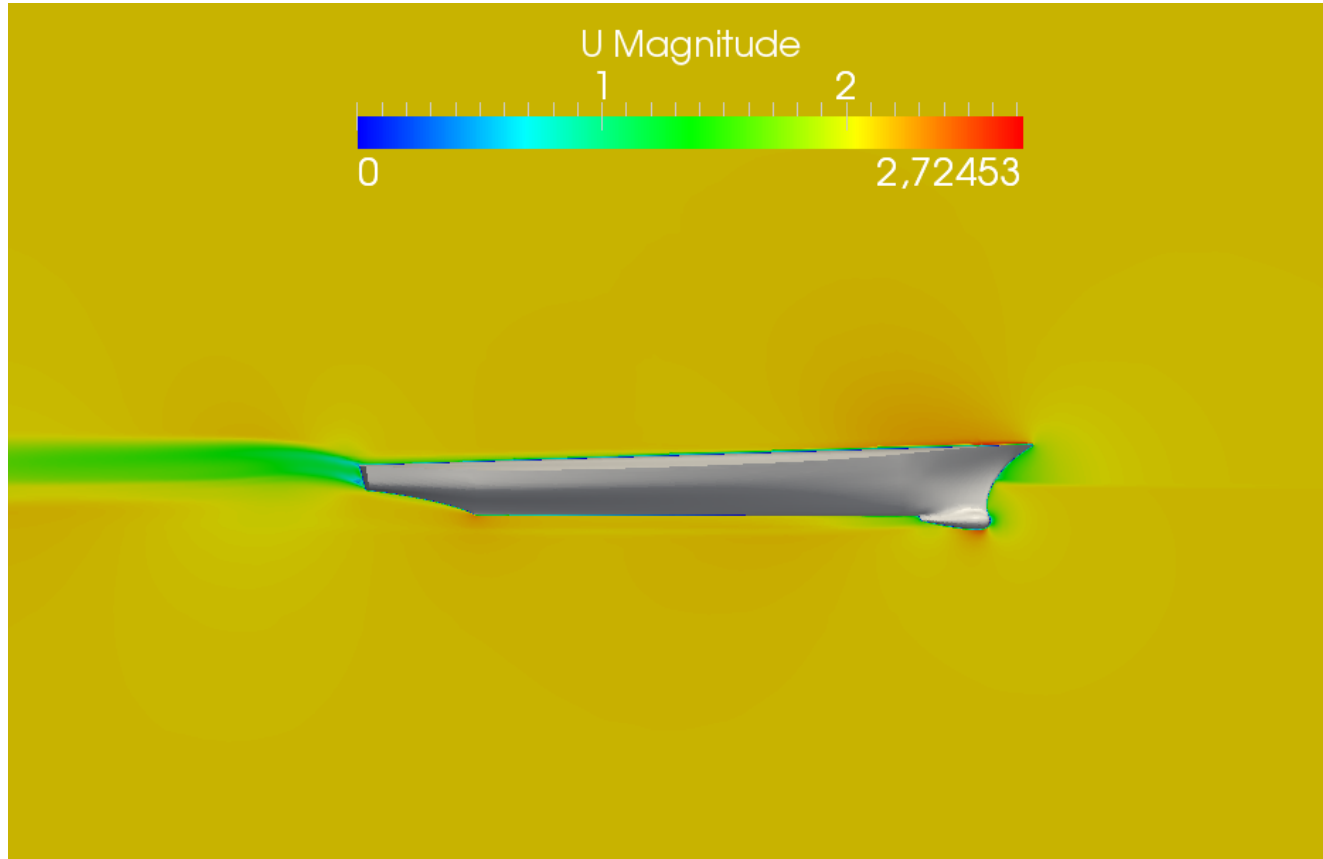


# OpenFOAM vs commercial CFD (GS)

- Mass fraction visualized on hull surface: wave shape detected
- Excellent agreement with *Gold-Standard* results

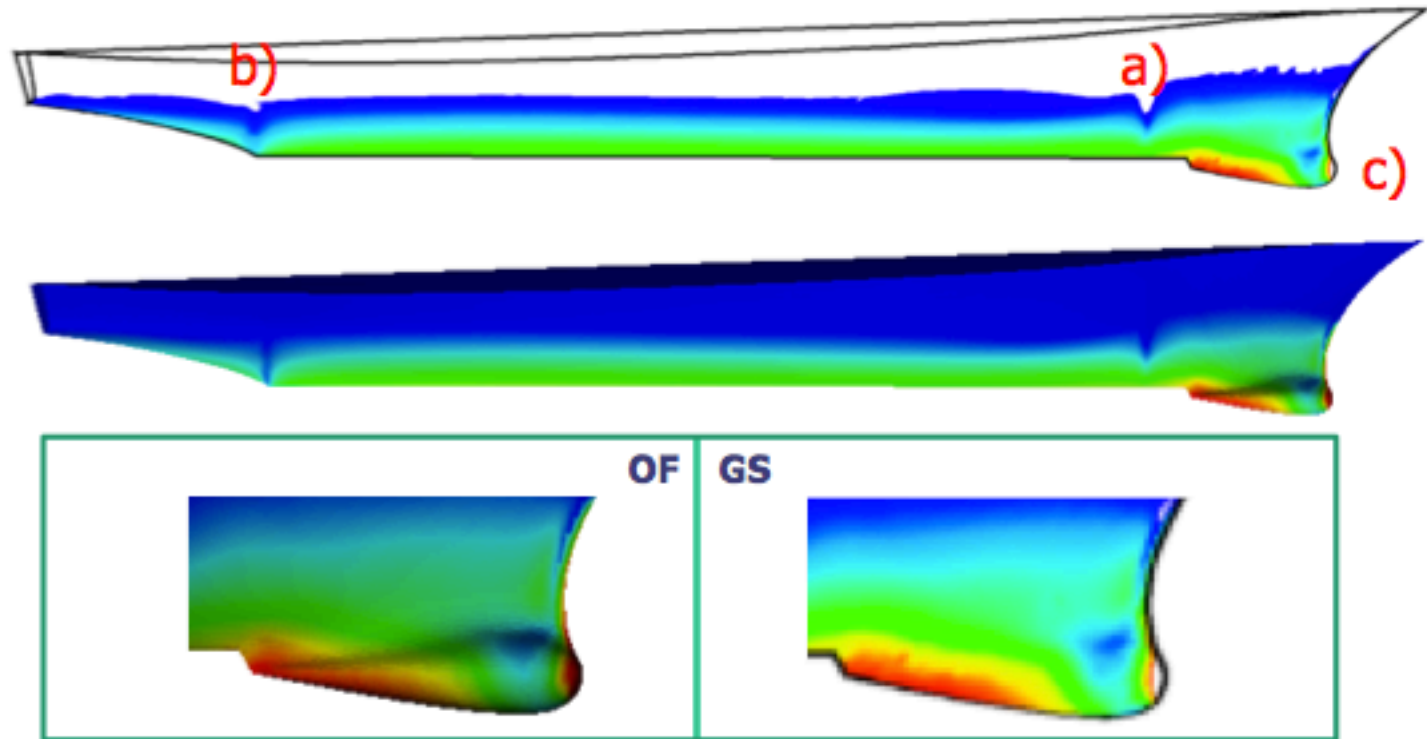


# OpenFOAM velocity field



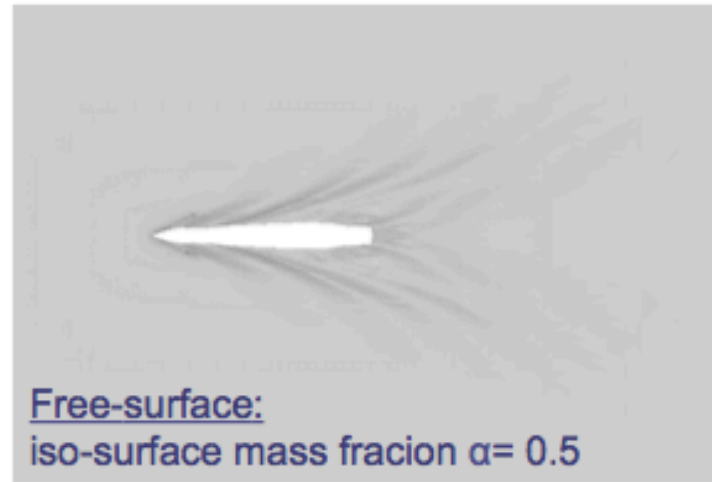
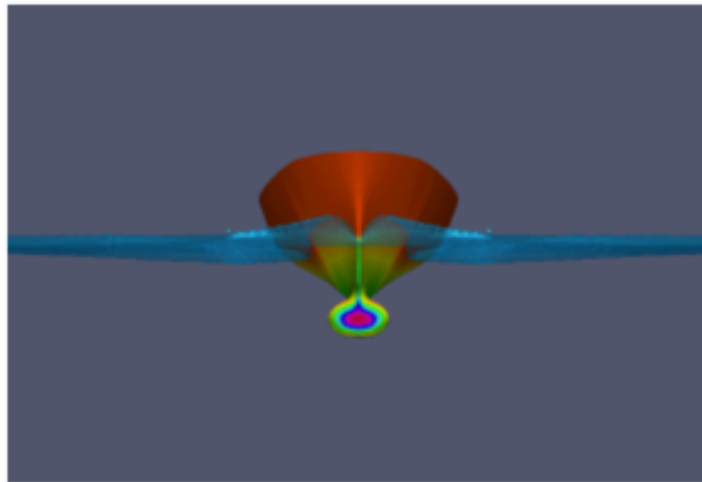
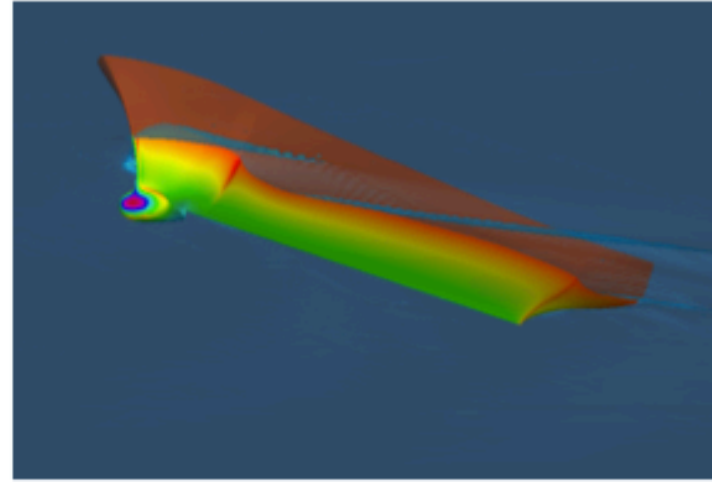
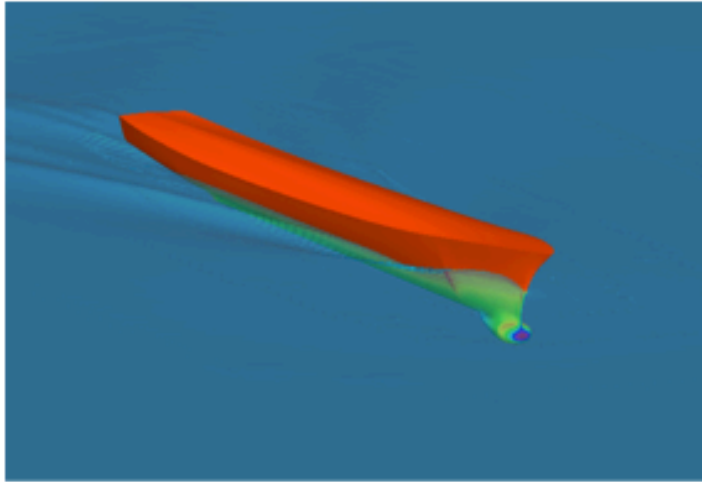
➤ Symmetry well caught by the solver in the velocity field computations

# Pressure field over hull



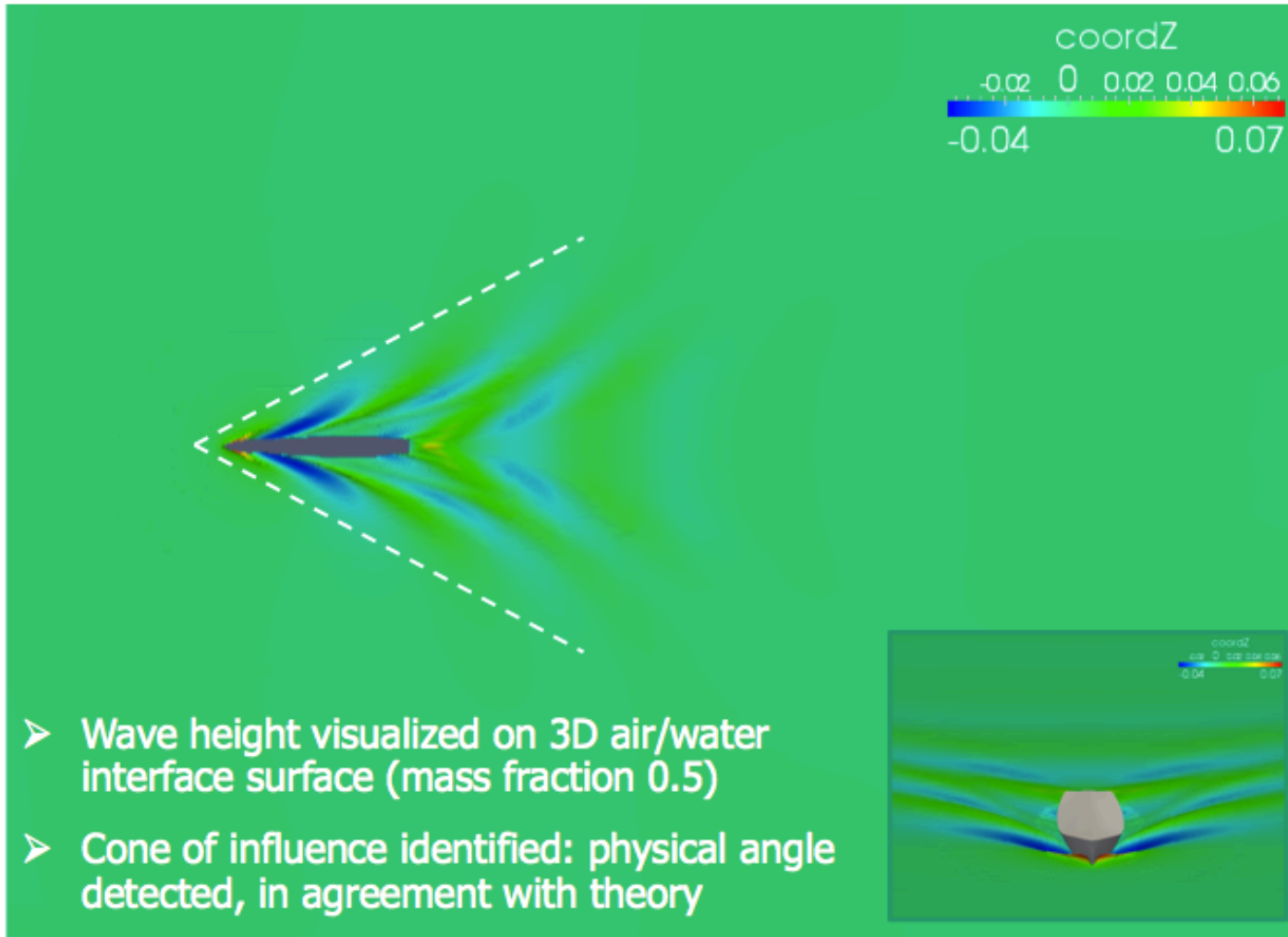
- a) Zero pressure distribution well caught
- b) Zero pressure transom well caught
- c) Bulb pressure distribution to be further investigated

# Free surface visualization

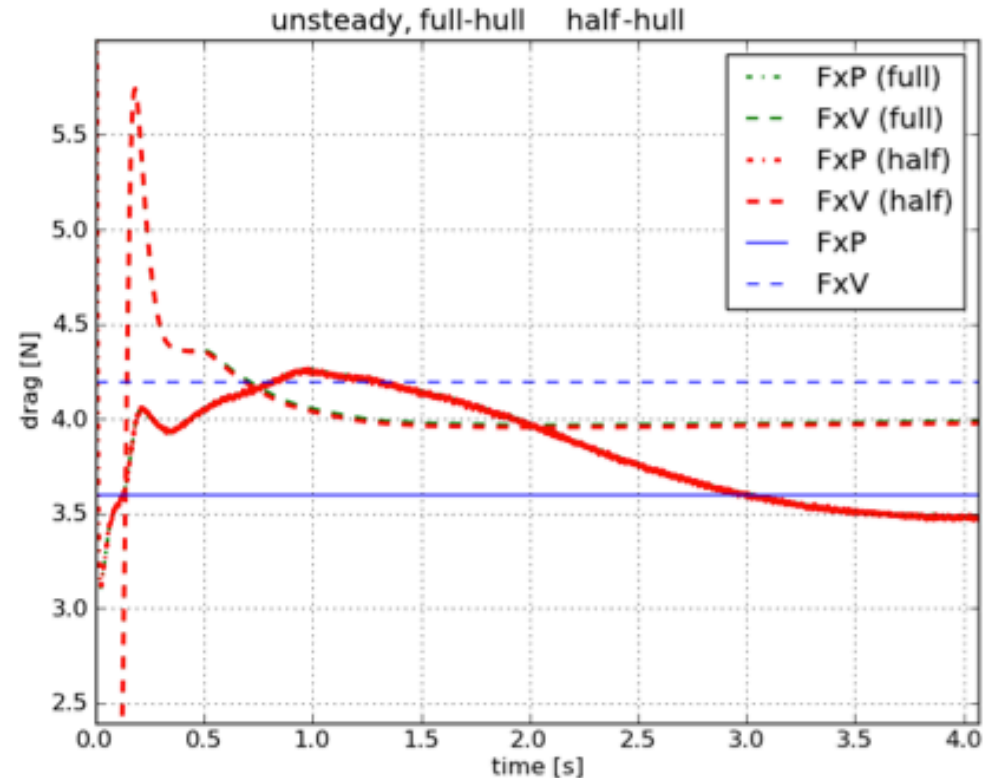
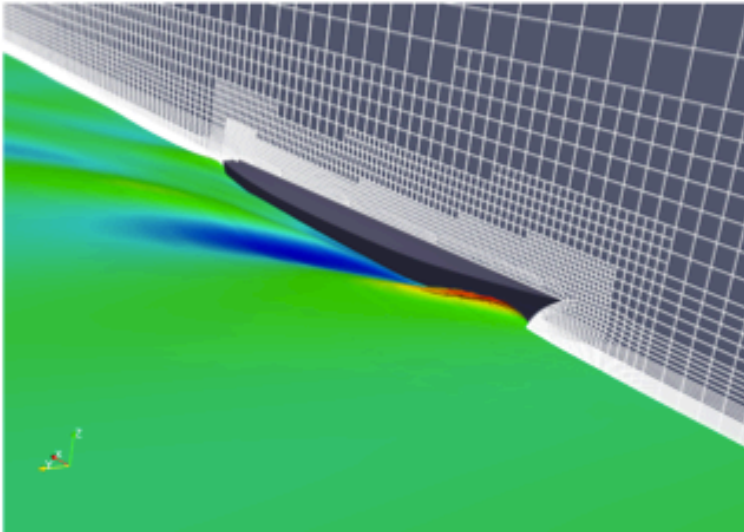


Free-surface:  
iso-surface mass fraction  $\alpha = 0.5$

# CFD results: agreement with theory

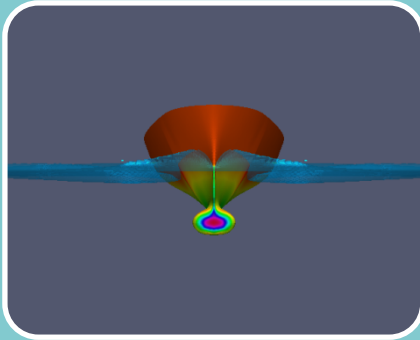


# DTMB-5414: half hull simulations



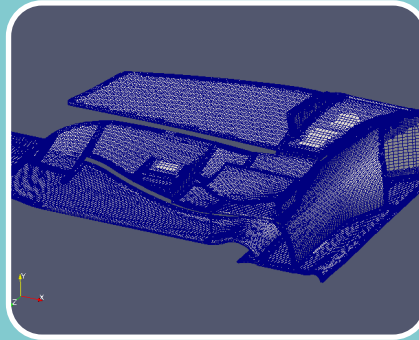
➤ save 43.5% computational time

# CFD skills applied to AC72 issues



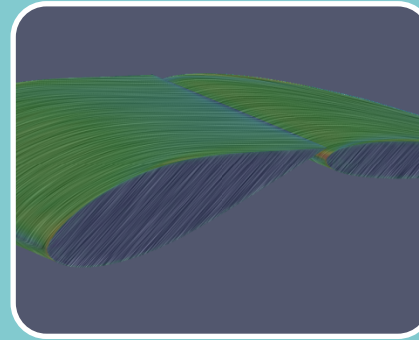
## Two phase High Reynolds RANS CFD analysis

Free surface simulation of high performance boat (AC72 kat) and appendages



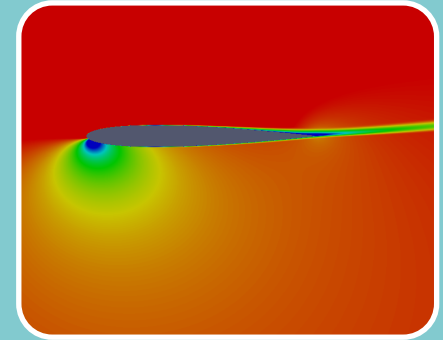
## 3D complex geometries meshing

Highly automated meshing process of 3D complex shapes; fully-structured, hybrid or unstructured mesh on problem demand.



## Aerodynamics

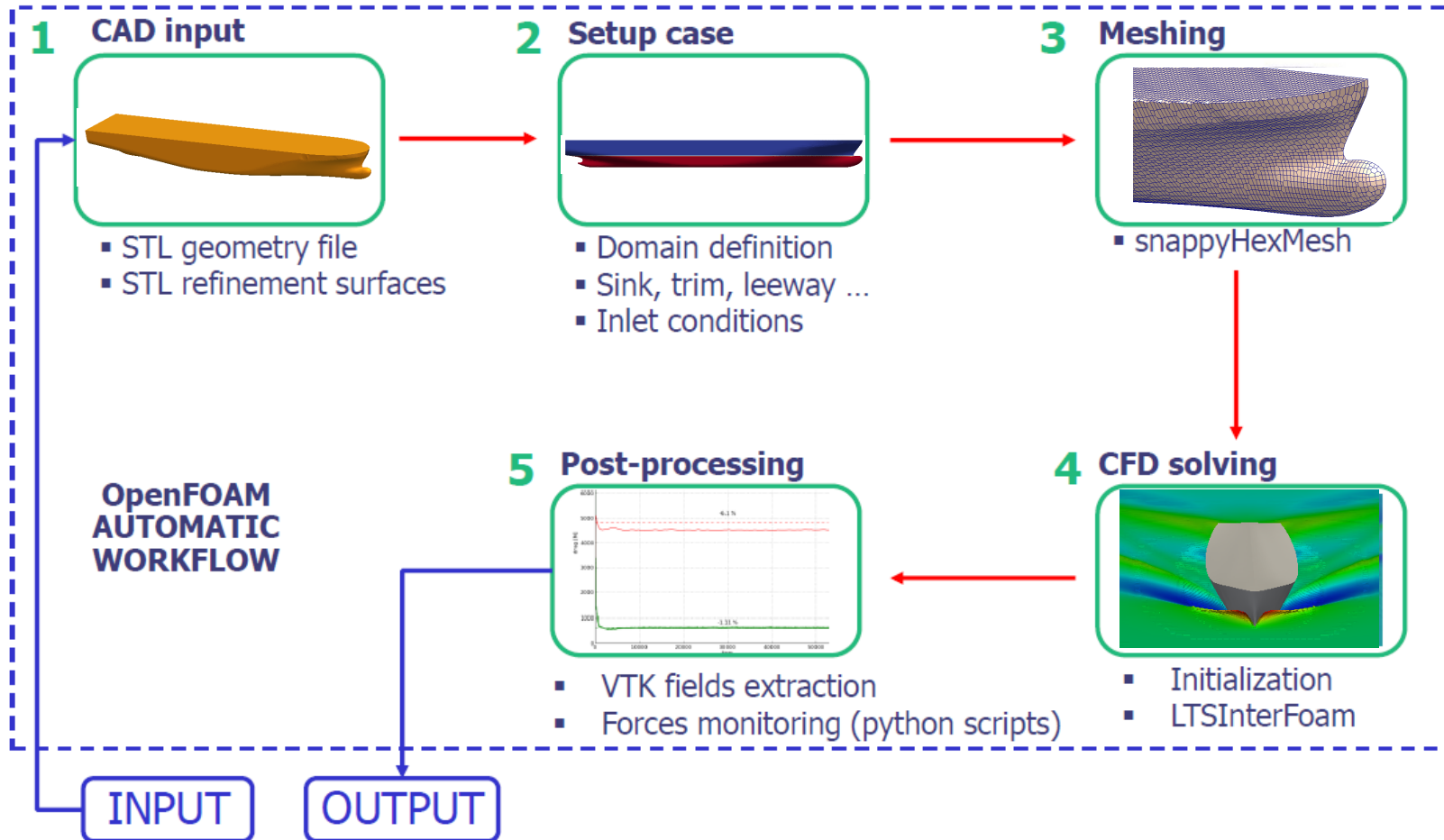
Aerodynamic of high Reynolds number RANS simulation of 3D bodies; high parallel CFD computations



## 2D airfoil design

Wing section efficient RANS simulation. Airfoil design optimization based on RANS code data

# Marine CFD automatic workflow





# OpenFOAM automatic workflow evaluation

## Automation

① Accuracy

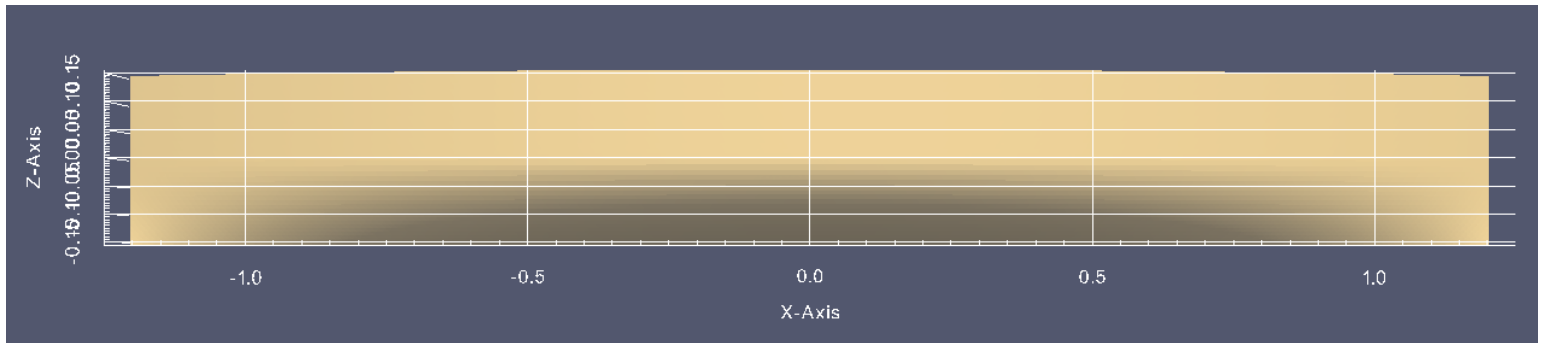
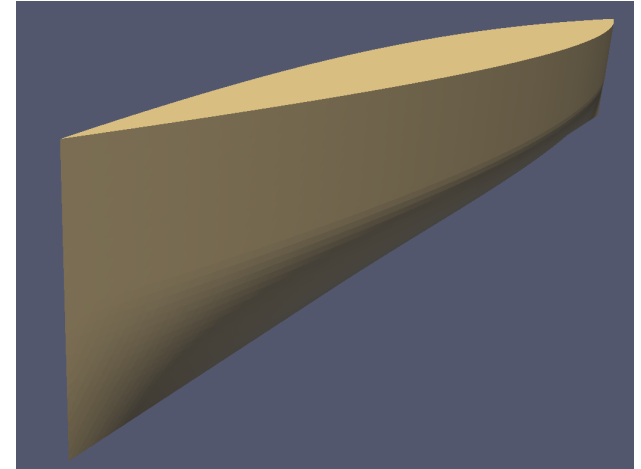
② Scalability

③ Reliability

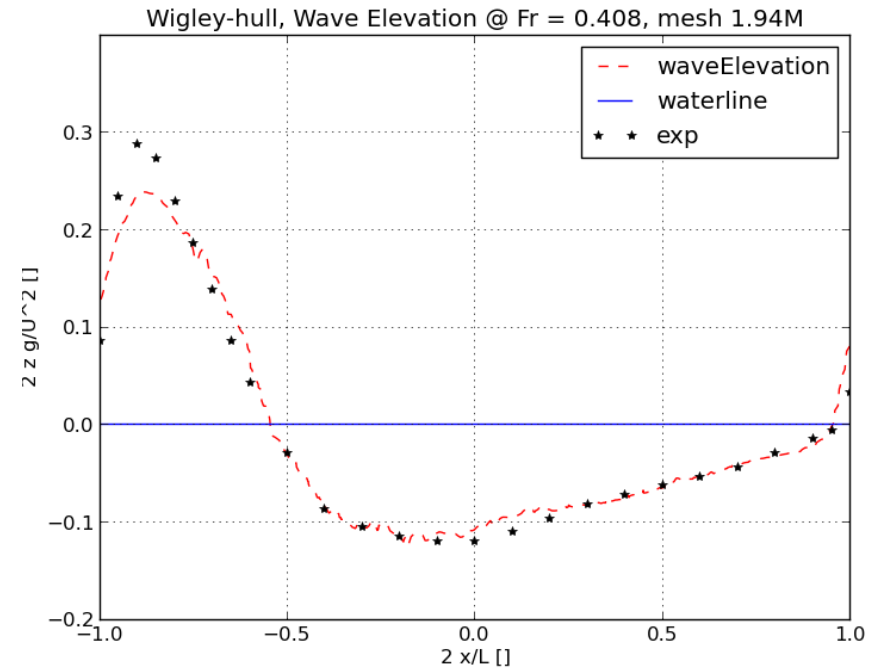
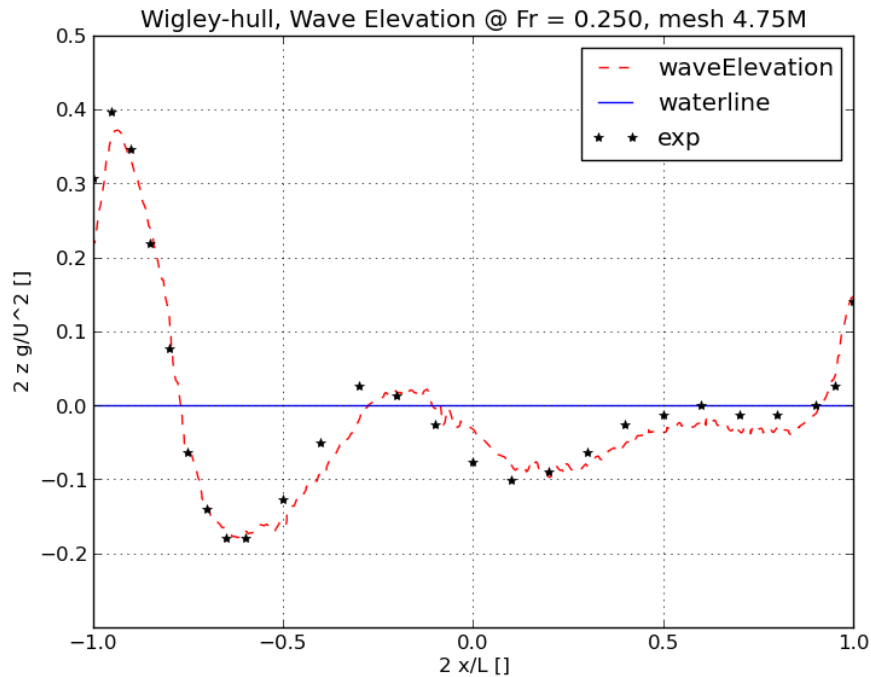
**Ready to CFD  
production on  
HPC cluster**

# Wigley-hull

- **Description:** widely used in marine engineering for validation of measures
- Standard reference



# Accuracy: CFD vs experimental



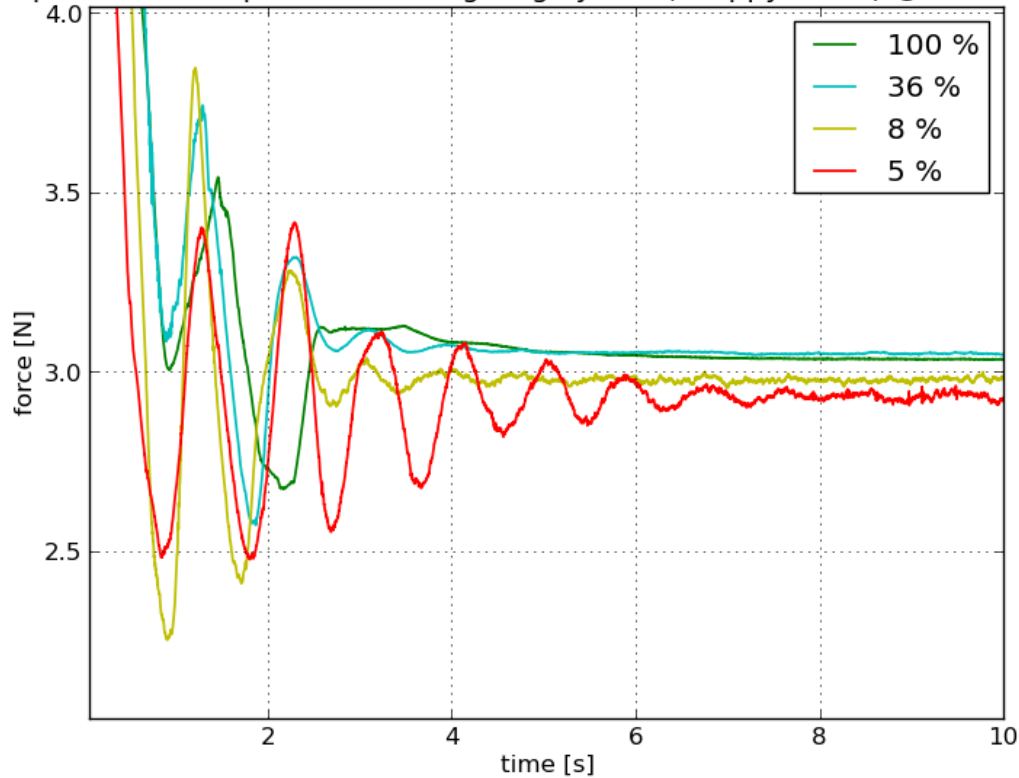
➤ Wigley-hull **wave elevation** @ different Froude number

# Accuracy: mesh sensitivity

- Fixed Froude number. On purpose degradation of mesh reducing number of cells to investigate how total computed forces become (in)accurate
- Considerable advantages in elapsed time required
- Mesh size range [% cells respect to gold-standard mesh]:  
5.0% - 8.0% - 36.% - **100.% (gold-standard)**
- Cores range: 12 – 24 @ PLX, CINECA cluster

# Accuracy: mesh sensitivity

OpenFOAM computed total drag: wigley-hull (snappy mesh) @ Fr = 0.250



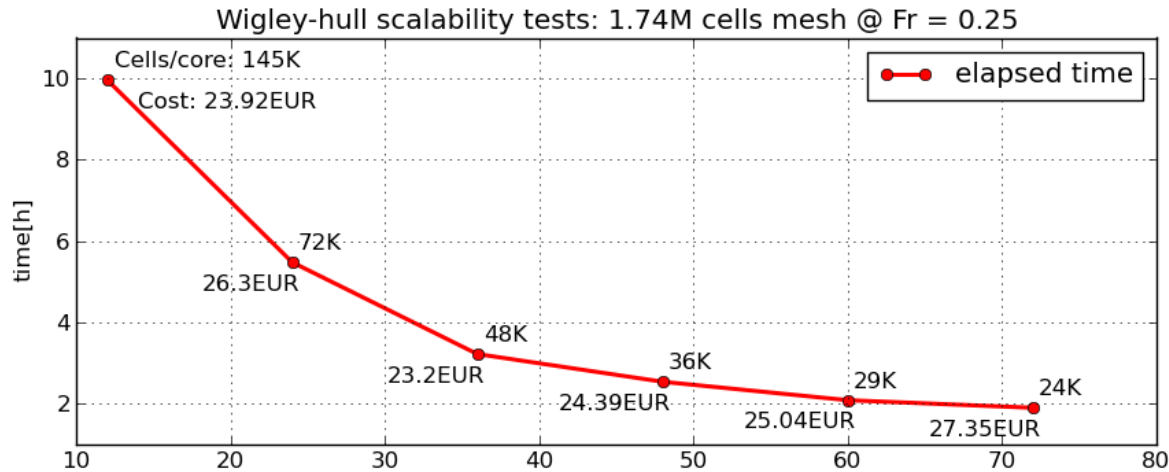
Mesh-size	F value [N]	F diff%
100 %	3,03	Used-as-GS
36 %	3,05	0,6%
8 %	2,98	1,6%
5 %	2,93	3,3%

- Reducing mesh size is not critical for the absolute convergence but just delays it.
- 5% size mesh respect to GS produces a 3% discrepancy in the total computed drag
- 5% size mesh respect to GS requires just 2h @ 12 cpu to reach convergence
- **User choice: different accuracy, different mesh size, different cost.**

# Scalability

- Different elapsed time due to different used computational cores
- Fixed **mesh** size: 1.7 M cells
- **Cores** range: 12 – 24 – 36 – 48 – 72 @ PLX, CINECA cluster
- Fixed number of iterations: **5000** (up to convergence)
- Key value **indices**: elapsed-time, speedup, efficiency

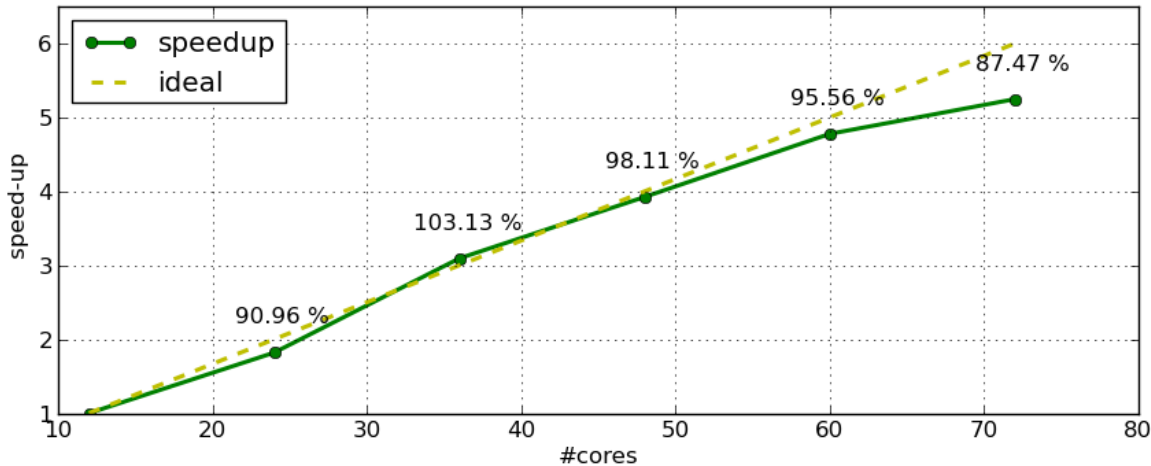
# Scalability results



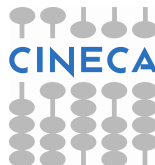
- Speed-up:

$$S = \frac{et_s}{et_p}$$

- Efficiency:

$$E = \frac{S}{cores}$$


- Convergence reached in **2h**
- High efficiency up to **24k** cells/core



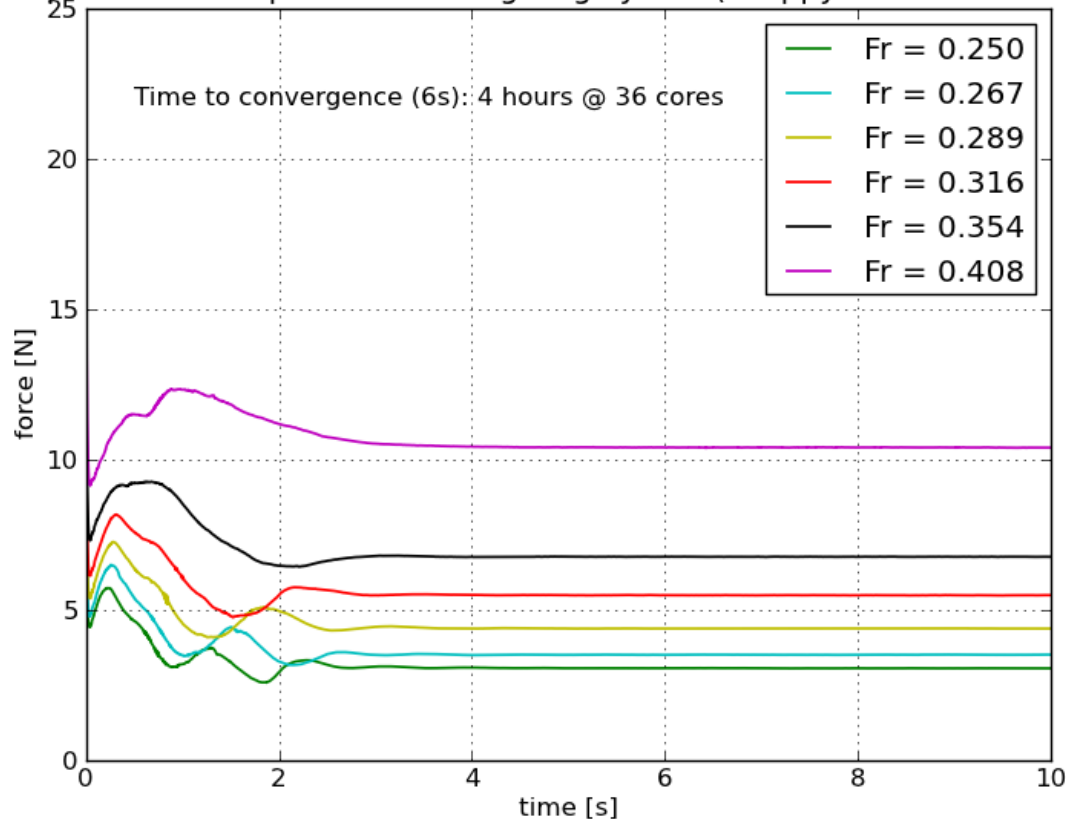
# Reliability

- Different computed forces due to different Froude number e.g. inlet velocity
- Fixed **mesh** size: 1.7 mln cells
- Fixed number of **cores**: 36 @ PLX, CINECA cluster
- **Froude** number range: 0.250 0.267 0.289 0.316 0.354 0.408
- Key value **indices**: total forces, viscous forces, pressure forces, wave height



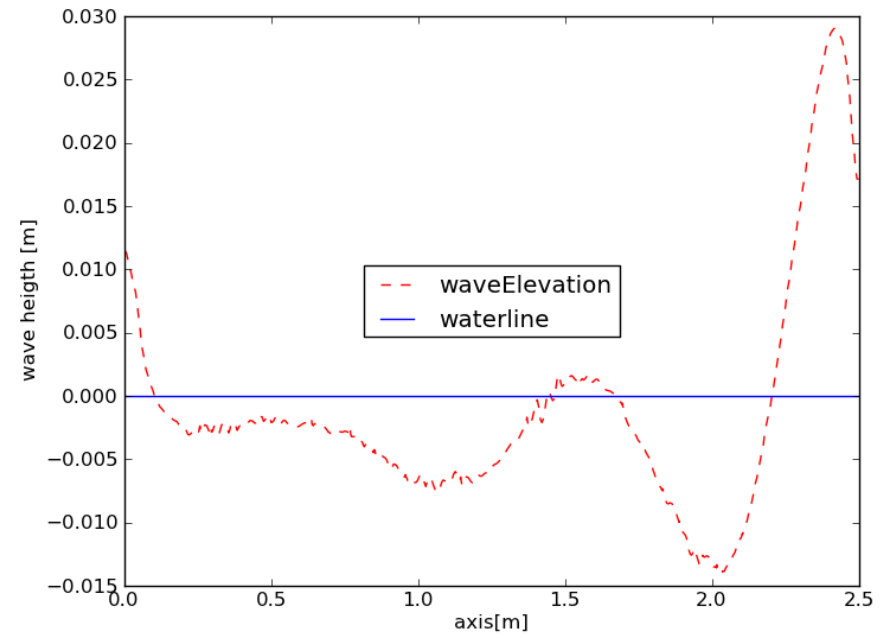
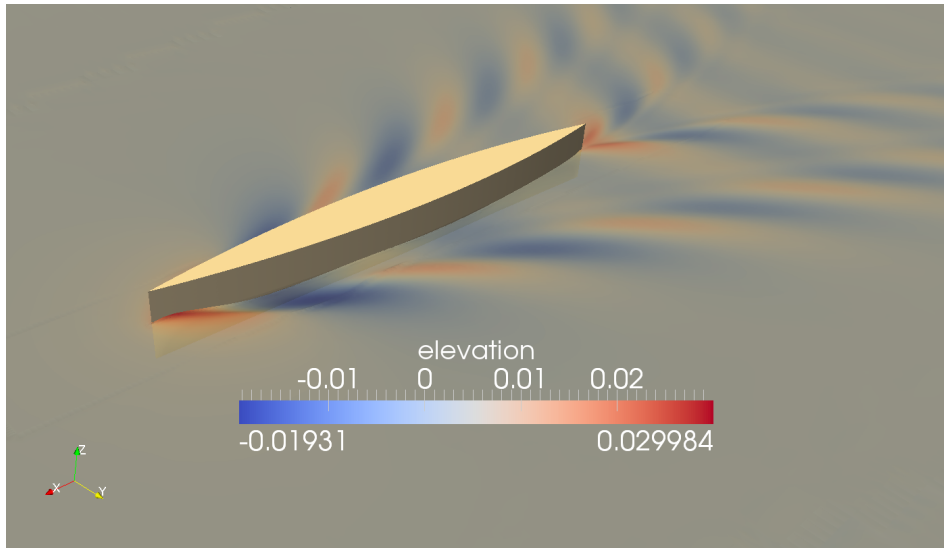
# Reliability: results

OpenFOAM computed total drag: wigley-hull (snappy mesh: 1.8M cells)

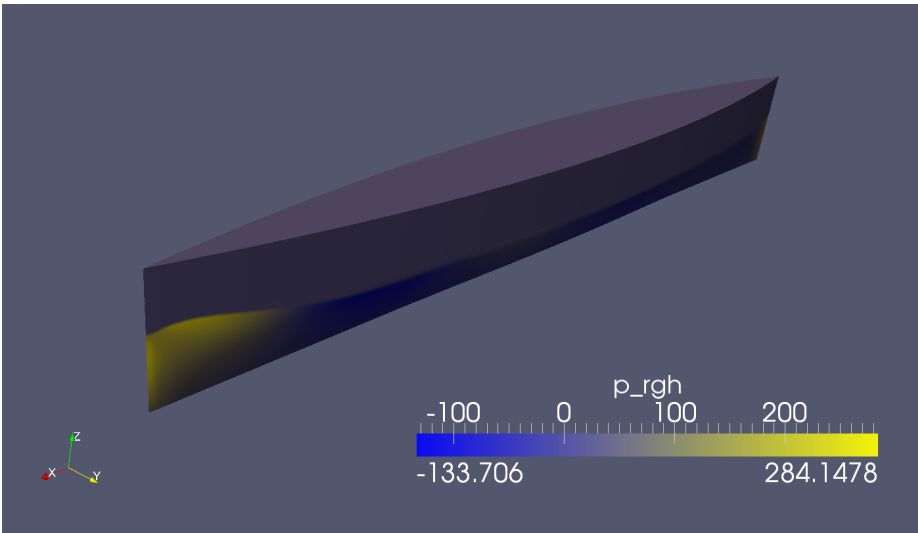


- Stable solution reached within **4s** (4k iteration for LTS solver)
- Fixed **cut-off** at 6s.
- Stable **means** are computed in the selected range 4s – 6s, so 4h @ 36 cpu exploiting best scalability

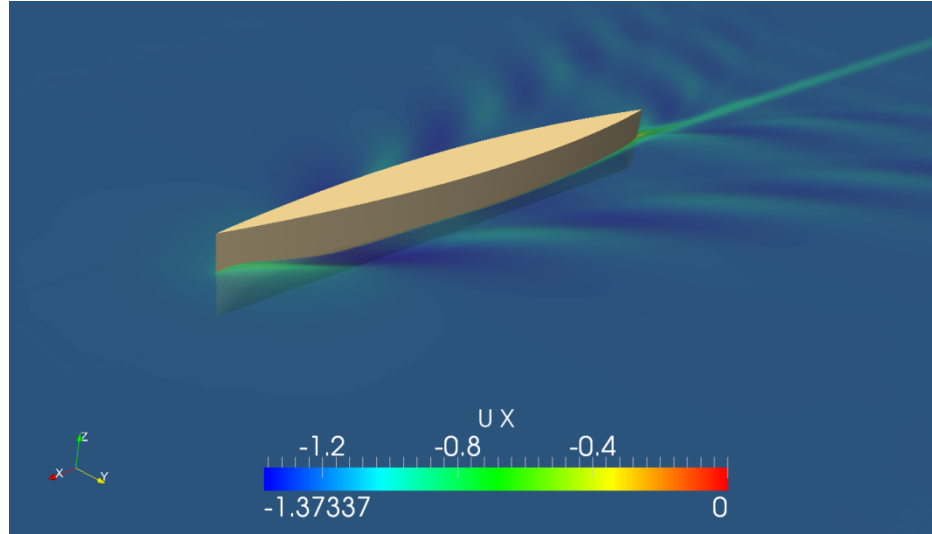
# Wave elevation ( $\alpha = 0.5$ )



# Pressure & axial velocity

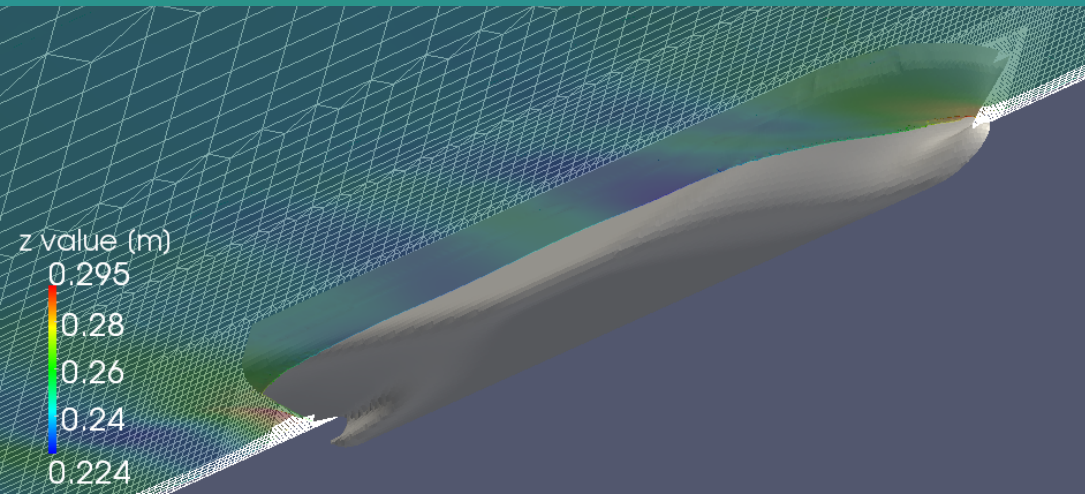
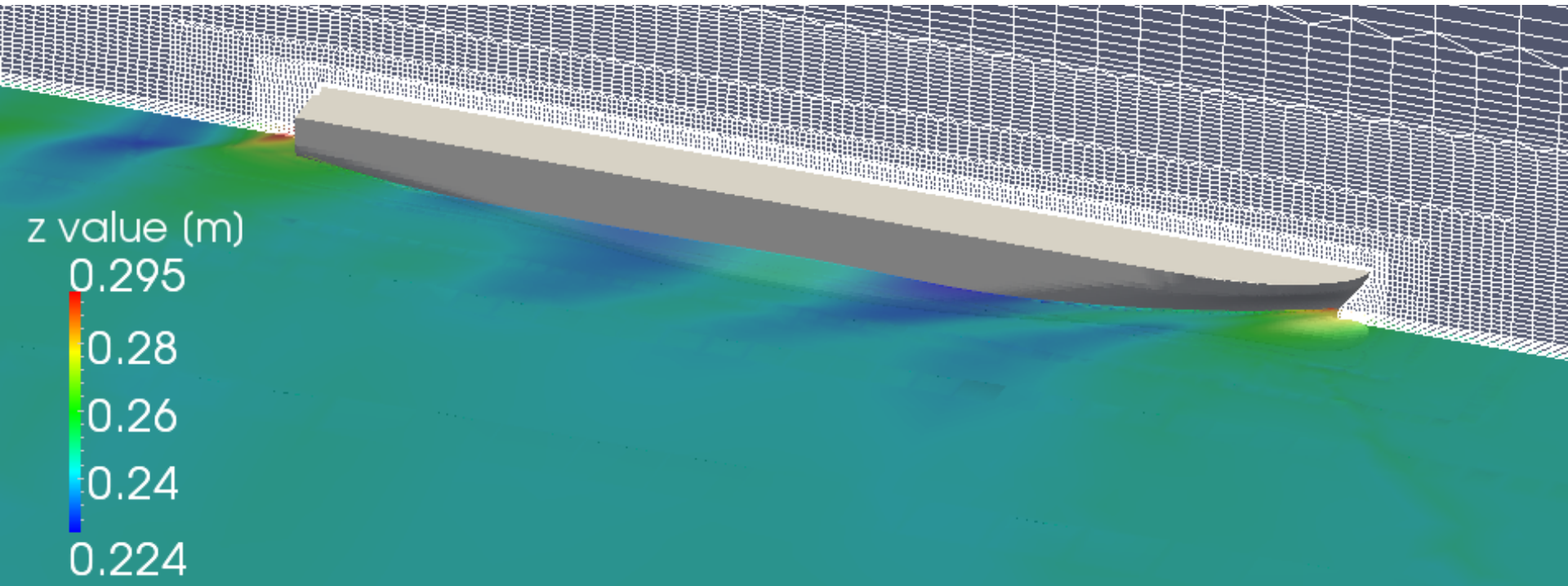


*Pressure over boat hull*



*Axial (x) velocity over wave 3D surface*

# Hands-on: CFD result



DTC Hull tutorial:

`$FOAM_TUTORIALS/multiphase/  
LTSInterFoam/DTCHull`



# Hands-on: OpenFOAM commands

## ① CAD transformation: scaling, trim, sink

- `surfaceTransformPoints -scale`  
`-yawPitchRoll`  
`-translate`

## ② Setup constants:

- Edit `constant/transportProperties`
- Edit `constant/RASProperties`

## ③ Setup BCs:

- Edit 0.org files

## ④ Setup free surface initial position:

- Edit `system/setFieldDict`
- Run `setFields`

## ⑤ Decompose domain:

- Edit `system/decomposeParDict`
- Run `decomposePar`

## ⑥ Run solver:

- `mpirun -np ... LTSInterFoam -parallel`

## ⑦ Reconstruct domain

- Run `reconstructPar`